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Stagnation point flow of Eyring Powell fluid in a vertical cylinder with heat transfer

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Abstract

In this paper an analysis is carried out to examine the effects of natural convection heat transfer for steady boundary layer flow of an Eyring Powell fluid flowing through a vertical circular cylinder. The governing partial differential equations along with the boundary conditions are reduced to dimensionless form by using the boundary layer approximation and applying suitable similarity transformations. The resulting nonlinear coupled system of ordinary differential equations subject to the appropriate boundary conditions is solved using the analytic technique homotopy analysis method (HAM). The effects of the physical parameters on the flow and heat transfer characteristics are presented. The behavior of skinfriction coefficient and Nusselt numbers are also studied for different parameters.

Keywords: Boundary layer flow; Vertical cylinder; Eyring Powell fluid; HAM

1. Introduction

The theory of mixed convection effects comprising temperature difference at different locations of the fluid and heat flow due to some external agent is used widely owing to its important applications in the world of industry and technology. Bachok et al [1] analyzed suction and injection effects on the problem of mixed convection boundary layer steady flow of a viscous fluid over a permeable vertical flat plate embedded in an anisotropic porous medium. In another work, Ahmad et al [2] examined the influence of temperature dependent variable viscosity over the flow of mixed convection boundary layer flow past an isothermal horizontal circular cylinder. Further, Cheng [3] studied the natural convection heat transfer of non-Newtonian fluids in a porous medium from a downward-pointing vertical cone under mixed thermal boundary conditions. Recently, Rashad et al [4] examined the influence of uniform transpiration velocity effects on free convective boundary layer

Eyring Powell fluid is a three constant fluid model that is capable of displaying a non-zero bounded viscosity at both the surface of the sheet and the fluid at infinity. Noreen and Nadeem [19] investigated the heat and mass transfer characteristics on the peristaltic flow of an Eyring Powell fluid in an endoscope. In another effort, Yurusoy [20] studied the problem of pressure distribution of a slider bearing lubricated with Eyring Powell fluid. The purpose of the present investigation is to examine the mixed convection heat transfer effects over the steady incompressible flow of Eyring Powell fluids over a vertical circular cylinder. To the best knowledge of the authors the stagnation flow of Eyring Powell fluid in a cylinder has not been explored to date. The solutions of the problem were produced using the homotopy analysis method (HAM). A comparison of the present

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flow of a non-Newtonian fluid over a permeable vertical cone embedded in a porous medium saturated with a nanofluid. In a recent work, Nadeem et al [5] studied the effects of mixed convection heat transfer for the boundary layer flow of a steady viscous nanofluid over a vertical slender cylinder. A few other interesting efforts concerning the concept of mixed convection heat transfer are included in [6-18].

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solutions is also presented as a special case with the work in [5]. The paper concludes with a results and discussion section that contains a detailed discussion about the physical features the problem entails.

2. Formulation

Consider the problem of mixed convection boundary layer flow of an Eyring Powell fluid through a vertical circular cylinder having radius a The temperature at the surface of the cylinder is assumed to be a constant T_w and the uniform ambient temperature is taken to be T_∞ such that the quantity $T_w-T_\infty>0$, in the case of the assisting flow, while $T_w-T_\infty<0$, in case of the opposing flow, respectively. Under the boundary layer assumptions the equations of motion and heat transfer are

$$(rw)_r + ru_x = 0, (1)$$

$$uu_{x} + wu_{r} = U\frac{dU}{dx} + v(1+M)(u_{rr} + \frac{1}{r}u_{r}) - \frac{2}{3\rho\beta c^{3}}$$

$$\left[\frac{w^{2}}{r^{2}}u_{rr} + u_{r}(\frac{2}{r^{2}}ww_{r} - \frac{1}{r^{3}}w + u_{x}u_{rx} - \frac{1}{r}wu_{rx} - w_{rr}u_{x} - w_{r}u_{rx} - \frac{1}{r}w_{r}u_{x}) - w_{r}u_{rr}u_{x} + \beta^{*}(T - T_{\infty})g\right],$$
(2)

$$wT_r + uT_x = \alpha (T_{rr} + \frac{1}{r}T_r) + \frac{\nu}{c_p} (1 + M)u_r^2 - \frac{4\gamma M}{3c_p c^2} u_r^2 (\frac{w^2}{r^2} - w_r u_x),$$
(3)

where the velocity components along the (x, r) axes are (w, u), ρ is density, ν is the kinematic viscosity, p is pressure, c and β are the material fluid parameters, β^* is the coefficient of thermal expansion, g is the gravitational acceleration in x-direction, M is the Eyring Powell parameter, T is the temperature, γ is the curvature parameter, α is the thermal diffusivity, c_p is the specific heat at constant pressure, U_∞ is the surface fluid velocity, and U is the free stream velocity and is defined as $U = U_\infty\left(\frac{x}{I}\right)$.

The corresponding boundary conditions for the problem are

$$u(x, a) = 0,$$
 $u(x, a) \rightarrow U(x) \text{ as } r \rightarrow \infty,$ (4)

$$T(x, a) = T_w(x), \quad T(x, a) \to T_\infty \text{ as } r \to \infty.$$
 (5)

3. Solution of the problem

Introduce the following similarity transformations [21]:

$$u = \frac{xU_{\infty}}{l}f'(\eta), \qquad w = -\frac{a}{r}\left(\frac{vU_{\infty}}{l}\right)^{\frac{1}{2}}f(\eta), \qquad (6)$$

$$\theta = \frac{T - T_{\infty}}{T_w - T_{\infty}}, \qquad \eta = \frac{r^2 - a^2}{2a} \left(\frac{U_{\infty}}{vl}\right)^{\frac{1}{2}}, \qquad (7)$$

where the characteristic temperature ΔT is calculated from the relations $T_w - T_\infty = \left(\frac{x}{l}\right)^2 \Delta T$. With the help of transformations (6) and (7), Eqs. (1) to (3) take the form

$$\begin{split} &(1+2\gamma\eta)\ \theta'' + 2\gamma\theta' + \Pr(f\theta' - 2f'\theta) + (1+M)\Pr{Ec}\ (1+2\gamma\eta)\ f^{''2} \\ &+ 2KM\ \Pr{Ec}[\gamma ff'f'' - (1+2\gamma\eta)\ f^{'2}f'' - \frac{\gamma^2}{(1+2\gamma\eta)}f^{'2}f''] = 0, \end{split} \tag{9}$$

in which $\gamma=\left(\nu l/U_{\infty}a^2\right)^{1/2}$, $M=1/\beta\mu c$, $K=2U_{\infty}^2/3c^2l^2$ is the dimensionless Eyring powell parameters, $\lambda=g\beta^*\Delta Tx/U_{\infty}^2$ is the buoyancy parameter, $\Pr=\nu/\alpha$ is the Prandtl number and $Ec=U_{\infty}^2/c_p\Delta T$ is the Eckert number.

The boundary conditions in nondimensional form are defined as

$$f(0) = b, \quad f'(0) = 0, \quad f' \to 1, \text{ as } \eta \to \infty,$$
 (10)

$$\theta(0) = 1, \quad \theta \to 0, \text{ as } \eta \to \infty,$$
 (11)

where b is any constant. The extra boundary condition of f(0) follows from the work in [21]. The important associated physical quantities such as shear stress at the surface τ_w , the surface heat flux q_w and the Nusselt numbers Nu are defined as

(4)
$$\tau_w = (\tau_{rx})_{r \to a}, \quad q_w = -k(\frac{\partial T}{\partial y})_{y=0}, \quad Nu/Re_x^{1/2} = -\theta'(0),$$

where τ_{rx} is the component of stress tensor, k is the thermal conductivity and Re_x is the local Reynolds number.

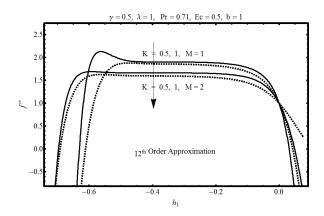


Figure 1: \hbar_1 —curves for f' plotted for different values of K and M

The solution of the present problem is obtained by using the powerful analytical technique homotopy analysis method (HAM). In the present case we seek initial guesses to be [19, 20, 22–30]

$$f_0(\eta) = b - 1 + \eta + e^{-\eta}, \quad \theta_0(\eta) = e^{-\eta}.$$
 (13)

The corresponding auxiliary linear operators are

$$L_f = \frac{d^3}{d\eta^3} + \frac{d^2}{d\eta^2}, \qquad L_\theta = \frac{d^2}{d\eta^2} + \frac{d}{d\eta},$$
 (14)

satisfying

$$L_f[c_1 + c_2\eta + c_3e^{-\eta}] = 0, \quad L_\theta[c_4 + c_5e^{-\eta}] = 0,$$
 (15)

where c_i (i = 1, ..., 5) are arbitrary constants. The zeroth-order deformation equations are

$$(1-q) L_f[\hat{f}(\eta; q) - f_0(\eta)] = _q H_f \hbar_1 N_f[\hat{f}(\eta; q)],$$
(16)

$$(1 - q) L_{\theta}[\hat{\theta}(\eta; q) - \theta_{0}(\eta)] = _qH_{\theta}\hbar_2N_{\theta}[\hat{\theta}(\eta; q)], (17)$$

where the auxiliary convergence parameters H_f and H_{θ} , both are taken to be $e^{-\eta}$.

Further details of the HAM procedure can be found in the listed references. However, the numerical results of the present solutions are presented in the proceeding section.

4. Results and discussion

The problem of steady incompressible flow of non-Newtonain Eyring Powell fluid flowing through a vertical

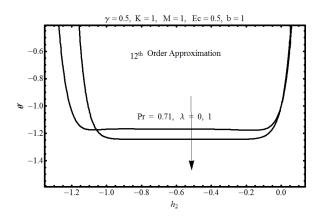


Figure 2: \hbar_2 —curves for θ plotted for different values of \Pr and λ

circular cylinder under the influence of mixed convection heat transfer and deceptions effects is computed by applying the powerful analytic technique homotopy analysis method (HAM). An associated fact with the HAM solutions is their dependence upon the auxiliary convergence parameters \hbar_1 and \hbar_2 corresponding to momentum and heat transfer, respectively. Figs. 1 and 2 are included to observe the convergence region of the involved auxiliary parameters for different combinations of other involved parameters. Fig. 1 shows the acceptable convergence regions computed at the surface of the sheet, for the auxiliary parameter h_1 for different combinations of K and M when the bouncy parameter $\lambda = 1$ and the deception rate Eckert number Ec = 0.5. It is noticed from Fig. 1 that with increase in K, the convergence region decreases, whereas the acceptable region for h_1 is greater for M=2 than for M = 1. The convergence region for velocity profile with $K = 1, M = 1 \text{ is } -0.5 \le h_1 \le -0.1.$ Fig. 2 is included to observe the convergence region for the auxiliary parameter \hbar_2 involved in heat transfer for different values of bouncy parameter λ , when Pr = 0.71. It is noted from Fig. 2 that the convergence region with $\lambda = 0$ is greater than for non-zero λ . Specifically for $\lambda = 0$ the convergence region is $-1.1 \le h_2 \le -0.1$.

The influence of different involved parameters over the heat and fluid flow are presented in Figs. 3 to 8. Figs. 3 to 5 are displaying the effects of some involved parameters over the fluid flow while Figs. 6 to 8 are drafted to observe the influence of different parameters over the heat transfer characteristics. Fig. 3 is prepared to predict the influence of curvature parameter γ and Eyring Powell fluid parameter M over the nondimensional velocity profile for presented values of the other involved parameters. From Fig. 3 it is observed that with increase in both γ and M the velocity profile decreases.

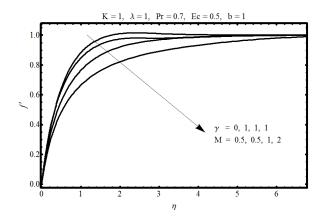


Figure 3: Influence of γ and M over f'

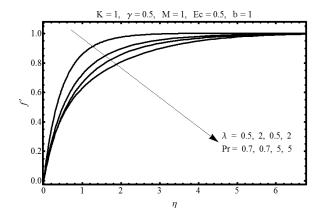
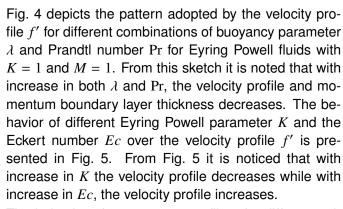


Figure 4: Influence of λ and \Pr over f'



The behavior of temperature profile θ for different values of Prandtl number \Pr and Eyring Powell parameter M is portrayed in Fig. 6. From Fig. 6 it is obvious that with increase in \Pr the temperature profile θ decreases while with increase in M the temperature profile θ increases. Fig. 7 gives the influence of curvature parameter γ and Prandtl number \Pr over the temperature profile θ . From Fig. 7 it is observed that with increase in both \Pr and γ the temperature profile decreases. Fig. 8 is included to observe the influence of K and Ec over the temperature profile θ . From this figure it is clear that

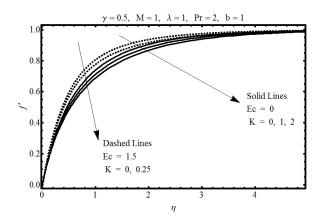


Figure 5: Influence of K and Ec over f'

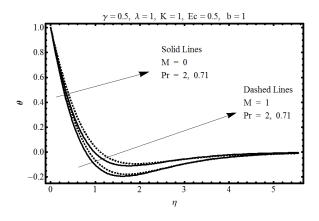


Figure 6: Influence of Pr and M over θ

with increase in K, the temperature profile θ decreases while with increase in Ec the temperature profile enhances.

Table 1 gives a comparison of the special case of the present solutions with the existing work [5]. From Table 1 it is noted that the two results are in excellent agreement. Table 2 contains the values of velocity boundary derivatives corresponding to the shear stress at the surface of the cylinder tabulated for different combinations of Eyring Powell parameters K and Eckert numbers Ec, whereas Table 3 is prepared for values of the temperature boundary derivatives corresponding

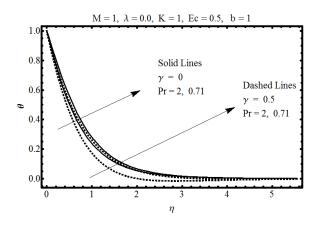


Figure 7: Influence of \Pr and γ over θ

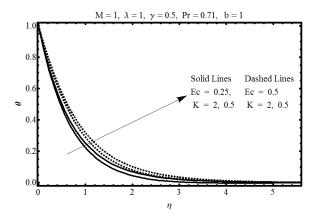


Figure 8: Influence of Ec and K over θ

to heat flux at the surface of the cylinder for different Prandtl numbers \Pr and \Pr Eyring Powell parameter M. From Table 2 it is noticed that with increase in Ec, shear stress at the surface increases for small K, while the surface shear stress decreases for larger K. From Table 3 it is deduced that with increase in \Pr heat flux at the surface of the cylinder increases whereas with increase in M heat flux at the surface of the cylinder decreases.

Table 1: Comparison of boundary derivatives for velocity profile of present results with [5] for various values of b and γ when $\lambda=0$ and M=0

	f"(0)							
	[5]	Present	[5]	Present	[5]	Present		
b\γ	0.5		1.0		1.5			
-1	0.9918	0.9918	1.1942	1.1942	1.3729	1.3729		
0	1.4886	1.4886	1.7244	1.7244	1.7954	1.7954		
1	2.0397	2.0397	2.1751	2.1751	2.2982	2.2982		
2	2.7332	2.7332	2.8029	2.8029	2.8746	2.8746		

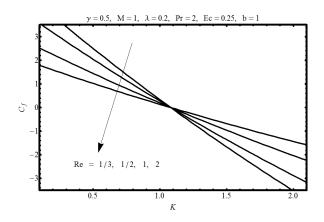


Figure 9: Influence of Re over c_f against K

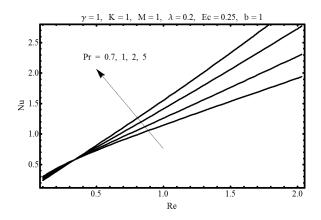


Figure 10: Influence of Pr over Nu against Re

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Table 2: Influence of boundary derivatives of velocity profile computed for different K and Ec when $\gamma=1/2,\ \lambda=1,\ M=1,\ Pr=2,\ b=1$

	f"(0)							
$K \backslash Ec$	0.0	0.5	1.0	2.0				
0.25 0.5 1.0 1.5 2.0	1.64972 1.64217 1.63028 1.62251 1.61867	1.70540 1.678150 1.62787 1.58337 1.54481	1.76175 1.71382 1.62154 1.53498 1.45576	1.87704 1.78499 1.59678 1.41169 1.23916				

Table 3: Influence of boundary derivatives for temperature profile computed for different Pr and M when $\gamma=1/2,\ \lambda=1,\ K=1,\ Ec=1/2,\ b=1$

	Θ '(0)								
Pr\M	0.0	0.2	0.5	1.0	1.5	2.0			
0.2	0.804943	0.797271	0.790688	0.786252	0.785742	0.76889			
0.7	1.41593	1.38826	1.36393	1.34629	1.34239	1.32229			
2.0	2.86071	2.77536	2.69642	2.63032	2.60349	2.5963			
5.0	5.40207	5.15247	4.90081	4.64499	4.48737	4.37883			
7.0	6.4818	6.09855	5.6955	5.25285	4.94765	4.71118			
10.0	7.17962	6.55968	5.87522	5.40583	5.18915	4.87365			

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